

Measurement of conductive heat transfer through rarefied binary gas mixtures

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Abstract

The conductive heat transfer through a rarefied binary gas mixture of helium and argon is experimentally investigated as a function of a molar fraction. The heat flux is measured from the free-molecular to near free-molecular flow regimes, and it is analyzed both in dimensional and dimensionless forms to clarify the effect from the gas-surface interaction. Then, the thermal accommodation coefficient is obtained by assuming a gas mixture as a single “virtual” species gas. The measured thermal accommodation coefficients for single component gases of helium and argon show good agreement with our previous results and data in literatures. The thermal accommodation coefficient for a binary gas mixture is compared with the theoretical value derived in the free-molecular flow regime as a superposition of an independent heat transfer by each gas component. The measured results are well explained by the theory. However, there is a little discrepancy between them, suggesting variation in the thermal accommodation coefficient with a molar fraction in flow regimes other than the free-molecular flow regime.

Key words: high Knudsen number flow, gas-surface interaction, thermal accommodation coefficient, molar fraction, heat flux

1. Introduction

2 The conductive heat transfer through a rarefied gas is a classical prob-
3 lem, and it has been studied by many researchers. In practical applications,

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4 gaseous mixtures are often employed, and the heat transfer through a gas
 5 mixture was studied extensively [1, 2, 3, 4, 5, 6, 7]. However, there are very
 6 few papers devoted to experimental measurements on this problem.

7 In a high Knudsen number flow, the number of collisions between gas
 8 and surface cannot be neglected compared with that between gas molecules.
 9 Then, gas-surface interaction is very important for understanding a thermal-
 10 fluid field in such a flow. Especially in a high Knudsen number flow of
 11 microscale, the large surface-to-volume ratio makes gas-surface interaction
 12 important. The amount of the conductive heat transfer is strongly affected
 13 by the gas-surface interaction. To represent gas-surface interactions in a
 14 practical purpose by a simple manner, an accommodation coefficient is often
 15 employed. It exhibits mean rate of exchanging a physical property between
 16 gas molecule and a surface without any detailed scattering behaviors. For
 17 a heat transfer problem, where the energy transfer between gas and surface
 18 is concerned, the energy accommodation coefficient α is employed, and it is
 19 defined by,

$$\alpha = \frac{\mathcal{E}_i - \mathcal{E}_r}{\mathcal{E}_i - \mathcal{E}_s}, \quad (1)$$

20 where \mathcal{E} is the energy flux of incident/reflected gas molecules with the indices
 21 i for incident molecules, r for reflected molecules, and s for molecules fully
 22 accommodated to a surface. If we focus on monatomic gas species, the kinetic
 23 energy of a gas molecule is concerned. The energy accommodation coefficient
 24 is also called as the thermal accommodation coefficient.

25 Thermal accommodation coefficients for single gas species have been mea-
 26 sured with various surface materials [8]. There are also several data on a gas
 27 mixture, like air. Theoretically, it sounds strange to consider the accommoda-
 28 tion coefficient for a gas mixture, because it is defined by molecular motions.
 29 It is obtained by assuming a gas mixture as a single component “virtual” gas,
 30 namely effective value. Nonetheless, it is useful in a practical simulation, like
 31 a study on a concentrated solar power plant [9, 10]. However, the variation
 32 of a thermal accommodation coefficient as a function of the mole fraction
 33 of components in a gas mixture is not well studied experimentally. Theo-
 34 retically, the thermal accommodation coefficient for gas mixture is expressed
 35 from the energy balance in the free-molecular flow regime as [11],

$$\alpha_{\text{mix}} = \frac{\sum \frac{X_i}{\sqrt{m_i}} \alpha_i}{\sum \frac{X_i}{\sqrt{m_i}}}, \quad (2)$$

36 where α_i is the thermal accommodation coefficient, χ_i is the mole fraction and
 37 m_i is the molecular mass, of the gas component i , respectively. This expres-
 38 sion is explained as follows: in the free-molecular flow regime, a conductive
 39 heat transfer through gas occurs by molecular motions, and the conductive
 40 heat flux q_{FM} for a single gas between two surfaces with a small temperature
 41 difference ΔT ($\ll T$) is expressed as [12, 13, 14],

$$q_{\text{FM}} = \frac{\alpha \gamma + 1}{8} \frac{\bar{v}}{\gamma - 1} p \Delta T, \quad \bar{v} = \sqrt{\frac{8kT}{\pi m}}, \quad (3)$$

42 where γ , \bar{v} , T , p , and m are the specific heat ratio, mean molecular speed,
 43 temperature, pressure, and molecular mass of the gas, respectively. The heat
 44 transfer for a gas mixture can be expressed by a superposition of heat transfer
 45 by each component, since there is no collision between gas molecules in gas
 46 phase in the free-molecular flow regime. Thus, the heat flux of a mixture
 47 q_{mix} is a sum of the heat flux of each component q_i with a weight of molar
 48 fraction χ_i ; i.e. $q_{\text{mix}} = \sum \chi_i q_i$. If components of a gas mixture has the same
 49 specific heat ratio γ , q_i is inversely proportional to the square root of the
 50 molecular mass of each component m_i , $q_i \propto 1/\sqrt{m_i}$ from Eq. (3), since T
 51 and p are common. It is important to note that the heat flux of mixture
 52 q_{mix} is calculated not from the mean molecular mass $m_{\text{mix}} = \sum \chi_i m_i$ [15],
 53 but from the mean molecular speed $\bar{v}_{\text{mix}} = \sum \chi_i \bar{v}_i$. Therefore, the thermal
 54 accommodation coefficient for a mixture is given by Eq. (2) for gas species
 55 with the same specific heat ratio. This theoretical equation was verified
 56 through the Nusselt number for various molar ratios of $\text{H}_2 - \text{N}_2$ mixture by
 57 Mikami, *et al.* [11]. However, the thermal accommodation coefficient of
 58 a mixture itself is not directly analyzed as a function of molar fraction of
 59 components in detail.

60 Recently, we measured the viscosity slip coefficient, which is related to the
 61 tangential momentum accommodation coefficient, for a binary gas mixture
 62 at several conditions of a molar fraction [16]. From the results, the viscosity
 63 slip coefficient deviated from estimated values from numerical studies with
 64 a constant tangential momentum accommodation coefficient of each compo-
 65 nent, suggesting that the tangential momentum accommodation coefficient is
 66 a concave function of a molar fraction in a binary gas mixture. Thus, it is not
 67 so surprising that the thermal accommodation coefficient deviates from the
 68 theoretical values in flow regimes other than the free-molecular flow regime.

69 In this study, the conductive heat transfer through a binary gas mixture
 70 of helium and argon is experimentally measured at many conditions of a mo-

71 lar fraction. Then, the thermal accommodation coefficients are obtained, and
72 they are compared with the theoretical expression, Eq. (2). Our experimen-
73 tal measurement system employs not only the free-molecular flow regime but
74 also the early transitional flow regime to deduce the thermal accommodation
75 coefficient, which is different from the theory, and the discrepancy between
76 our results and the theoretical value is discussed.

77 2. Measurements

78 2.1. Method

79 The heat flux was measured between two surfaces kept at different tem-
80 peratures through a rarefied gas mixture of helium and argon. The heat
81 transfer between a spherical vacuum chamber and a flat-shaped heater kept
82 at its center was measured as a function of pressure, and it was analyzed by
83 assuming a concentric spherical shells geometry.

84 The heat transfer rate from the heater to the chamber was measured
85 by the electrical consumption at the heater to keep its electrical resistance,
86 namely temperature, constant. The heat transfer consists of the heat con-
87 duction through gas confined in the chamber, the radiation, and the heat
88 conduction through the electrical leads of the heater. Only the first part
89 depends on pressure, and the latter two parts are independent of pressure;
90 thus, they are evaluated by the value in the vacuum limit, $p \rightarrow 0$ and are sub-
91 tracted from the calculated value. The heat flux was measured in a rarefied
92 condition, and convection was negligible.

93 The heat flux as a function of pressure was employed to derive the thermal
94 accommodation coefficient at the sample surfaces. In the free-molecular flow
95 regime, the heat flux from a heated surface at temperature T_H surrounded
96 by a monatomic gas in a chamber at temperature T_C is expressed by Eq.
97 (3), as previously explained. By designing the surface area ratio of T_H to T_C
98 very small, gas molecules undergo a large number of collision to the surface
99 at T_C before colliding to the surface at T_H , the temperature of the gas T in
100 Eq. (3) can be approximated by T_C . Then, the accommodation coefficient
101 α is derived from the heat fluxes measured at different pressure conditions
102 by fitting with Eq. (3). Again, for a gas mixture, the mean molecular speed
103 the mean molecular speed $\bar{v}_{\text{mix}} = \sum \chi_i \bar{v}_i$ was used. It is important to note
104 that the heat flux is independent of the geometrical configuration in the
105 free-molecular flow regime.

106 However, as explained in our previous study [14], it is not easy to accu-
 107 rately measure several data points in the free-molecular flow regime, *i.e.* at a
 108 high vacuum condition, in a simplified low-cost apparatus owing to unavoi-
 109 dable leakage and infinitesimal heat flux to measure. Therefore, we employed
 110 a general model expression to describe the heat flux in whole flow regimes,
 111 *i.e.* from the free-molecular to continuum flow regimes. The heat flux q
 112 can be modeled by a simple empirical interpolation of the value in the free-
 113 molecular limit q_{FM} , Eq. (3), and in the continuum limit, q_C , as in Refs.
 114 [13, 14, 17, 18, 19, 20], and it is expressed as

$$\frac{1}{q} = \frac{1}{q_{FM}} + \frac{1}{q_C}. \quad (4)$$

115 Recently, we analyzed the heat transfer in our measurement system [21],
 116 and we found that our system was well explained by a concentric spherical
 117 shells geometry. However, the heat flux started to deviate in high pressure
 118 conditions, and we proposed a revised empirical function for better fitting
 119 using numerical results by the S-model kinetic equation. Unfortunately, the
 120 revised function was derived only from the results of pure gases, helium,
 121 argon and xenon, and there is no reason for the function to be applicable to
 122 a gas mixture. Thus, we used the original form, Eq. (4), employed in our
 123 previous experimental studies [14, 13]. Meanwhile, pressure conditions were
 124 limited below 1.4Pa in the measurement to ensure the near free-molecular
 125 flow regime, so that the effect from the value in the continuum limit was
 126 minimized.

127 To use Eq. (4) for the fitting to obtain the accommodation coefficient, the
 128 heat flux in the continuum limit q_C is needed. The heat flux in the continuum
 129 limit q_C is described by Fourier's law, and it is independent of pressure but
 130 dependent on the geometrical configuration of two surfaces at different tem-
 131 peratures. Therefore, the geometry of a measurement system is better to be
 132 symmetry, and we design our system in the concentric spherical shells geom-
 133 etry. Considering that a sample with a flat surface is much easy to prepare,
 134 the flat-plate shaped heater with sample substrates is employed instead of an
 135 inner spherical shell. By making the size of the heater much smaller than the
 136 external spherical shell, *i.e.* the spherical vacuum chamber, the heat transfer
 137 problem between the heater and the chamber can be approximated by the
 138 problem in the concentric spherical shells geometry [21, 14]. The equivalent
 139 radius of the inner spherical shell calculated from a sphere having the same
 140 surface area with the heated surfaces was obtained as $R_H = 4.95\text{mm}$. The

141 radius ratio of the concentric spherical shells geometry, $\mathcal{R} = R_C/R_H$, was
 142 equal to 10. By assuming the geometry, the theoretical heat flux in the con-
 143 tinuum limit q_C could be obtained. The temperature distribution and the
 144 temperature dependence of the thermal conductivity in the continuum limit
 145 are considered following our recent analysis [21]. The temperature variation
 146 from the conductive heat transfer between two concentric spherical shells in
 147 radius direction r is described from the energy balance for a static fluid as,

$$\frac{\partial}{\partial r} \left(r^2 \kappa(T) \frac{\partial T}{\partial r} \right) = 0, \quad (5)$$

148 where $\kappa(T)$ is the thermal conductivity of the gas, which is a function of
 149 temperature. The heat flux is calculated by the Fourier's law as,

$$q = -\kappa(T) \frac{dT}{dr}. \quad (6)$$

150 The thermal conductivity $\kappa(T)$ was assumed to be proportional to T^{ω_T} in
 151 this study, which is similar to the model by using the inverse power law
 152 potential for a monatomic single gas, like the Variable Hard Sphere (VHS)
 153 model [22]. ω_T is named as the thermal conductivity index hereafter. Since
 154 the temperature of the gas in the vicinity of a wall is equal to that of the
 155 wall in the continuum limit, Eqs. (5) and (6) are solved analytically, and the
 156 heat flux from the heated inner spherical shell becomes [21],

$$q_C = \kappa(T_C) \frac{\mathcal{T}^{\omega_T+1} - 1}{(\omega_T + 1)(\mathcal{T} - 1)} (T_H - T_C) \frac{R_C R_H}{R_C - R_H} \frac{1}{R_H^2}. \quad (7)$$

157 where \mathcal{T} is the temperature ratio $\mathcal{T} = T_H/T_C$. The difference from the
 158 constant thermal conductivity case in our previous studies [14, 13] is that
 159 the heat flux is multiplied by a factor $(\mathcal{T}^{\omega_T+1} - 1) / \{(\omega_T + 1)(\mathcal{T} - 1)\}$.

160 The thermal conductivity of pure gas components, helium and argon, is
 161 determined by fitting the data in Ref. [23] by,

$$\kappa(T) = \kappa_{\text{ref}} \left(\frac{T}{T_{\text{ref}}} \right)^{\omega_T}, \quad (8)$$

162 where κ_{ref} is a reference value at temperature T_{ref} . The reference tempera-
 163 ture was chosen to be close to T_C . The thermal conductivity index ω_T was
 164 obtained from the gradient of a log-log plot of the thermal conductivity as a

Table 1: The reference thermal conductivity κ_{ref} at a reference temperature $T_{\text{ref}} = 300\text{K}$ and the thermal conductivity index ω_T for He-Ar binary gas mixture at every 10% of molar fractions.

He	Ar	κ_{ref} (mW/(m · K))	ω_T
0%	100%	17.84	0.8218
10%	90%	23.80	0.8064
20%	80%	30.57	0.7945
30%	70%	38.33	0.7842
40%	60%	47.30	0.7745
50%	50%	57.80	0.7648
60%	40%	70.23	0.7544
70%	30%	85.21	0.7428
80%	20%	103.6	0.7293
90%	10%	126.5	0.7130
100%	0%	156.0	0.6923

165 function of temperature. κ_{ref} at $T_{\text{ref}} = 300\text{K}$ and ω_T were 156.0mW/(m · K)
166 and 0.6923 for helium and 17.84mW/(m · K) and 0.8218 for argon, respec-
167 tively. To express the thermal conductivity for a gas mixture, the expression
168 by Mason and Saxena [24] was employed from the value for each component
169 as,

$$\kappa_{\text{mix}} = \frac{\kappa_1}{1 + A_{12} \frac{\chi_2}{\chi_1}} + \frac{\kappa_2}{1 + A_{21} \frac{\chi_1}{\chi_2}}, \quad A_{ij} = 1.065 \frac{\left[1 + \left(\frac{\kappa_i}{\kappa_j}\right)^{\frac{1}{2}} \left(\frac{M_i}{M_j}\right)^{\frac{1}{4}}\right]^2}{\left[8 \left(1 + \frac{M_i}{M_j}\right)\right]^{\frac{1}{2}}}, \quad (9)$$

170 where χ_i , M_i is the molar fraction and molecular weight of i component,
171 respectively. The molar fraction was varied every 10%, and the obtained
172 reference thermal conductivities and thermal conductivity indexes for the bi-
173 nary gas mixture are listed in Table 1. Although the effect of the continuum
174 limit was reduced by limiting pressure range, the effect from the thermal
175 conductivity is evaluated by using values obtained from recent *ab initio* sim-
176 ulations [25, 26, 27] and values in 1984 [28], which are available only for
177 limited molar ratio conditions.

178 2.2. Setup

179 The experimental setup is explained in detail elsewhere [14].

180 A spherical vacuum chamber made by Pyrex has the inner radius of $R_C =$
 181 49.5mm. It was immersed in a water bath at a room temperature to keep the
 182 temperature T_C constant. The pressure, which was assumed to be uniform
 183 inside the chamber, was measured by a temperature-controlled capacitance
 184 manometer (MKS, Baratron ®627B) directly connected to the chamber.

185 A tiny flat-shaped heater of $11.8 \times 12 \times 0.38 \text{ mm}^3$ designed for this system
 186 (Toyo Precision Parts MFG) was placed at the center of the spherical vacuum
 187 chamber. Two thin sample substrates were placed on both sides of the heater.
 188 For a surface material, platinum was selected, since there are many data and
 189 our previous measurements to check the validity of this measurement. A
 190 platinum foil with a thickness of $10 \text{ }\mu\text{m}$ (Nilaco) was employed. The heater
 191 was maintained at temperature T_H by an analog electrical bridge circuit, and
 192 the heat transfer rate from the heater was calculated from the voltage and
 193 current of the heater.

194 The test gas of a binary gas mixture was prepared by using a gas mixing
 195 system (Alicat Scientific, Flow Vision MX TM) with mass flow controllers
 196 (Alicat Scientific, MCV-100SCCM-D) from pure gas components of helium
 197 and argon prepared by commercially available gas cylinders.

198 3. Results and Discussion

199 3.1. Heat flux through binary gas mixture

200 The heat fluxes were measured three times by varying the pressure condi-
 201 tion for each molar ratio. The measured heat fluxes as a function of pressure
 202 are plotted in Fig. 1 (a). To show the scatterings of data as an indication of
 203 the measurement accuracy, all the measured heat flux are plotted. From the
 204 figure, the plots show similar profiles for all molar ratio conditions, and they
 205 increase almost linearly. However, if we focus on the results as a function of
 206 molar fraction, they are in a complicated order.

207 To make the problem simple and to be comparable to theoretical studies,
 208 the heat flux was evaluated in a dimensionless form. Following the previous
 209 studies [4, 29, 21], the rarefaction parameter δ_0 , which is inversely propor-
 210 tional to the Knudsen number, is defined as,

$$\delta_0 = \frac{R_0}{l}, l = \frac{\mu(T_0)v_0}{p}, v_0 = \sqrt{\frac{2kT_0}{m_0}}. \quad (10)$$

211 Here $R_0 = R_C - R_H$ is the characteristic length, l_0 is the equivalent mean free
 212 path, p is pressure inside the chamber, $\mu(T_0)$ and v_0 are the viscosity and the

213 most probable molecular speed at the reference temperature T_0 , respectively.
 214 The reference temperature T_0 is taken as $T_0 = T_C$. The viscosity for a mixture
 215 is calculated from the data for pure gases [23] by the Wilke's method. The
 216 discrepancies of δ_0 for a mixture (He 50%:Ar 50%) from that calculated with
 217 the viscosity obtained by using *ab initio* intermolecular potentials [27] were
 218 less than 3.4%. The reference molecular mass m_0 was calculated by the mean
 219 molecular mass of the mixture as,

$$m_0 = \chi_1 m_1 + \chi_2 m_2, \chi_1 + \chi_2 = 1. \quad (11)$$

220 The heat flux q is related to the dimensionless form q^* as,

$$q = q^* \cdot p v_0 \frac{\Delta T}{T_0}, \quad (12)$$

221 where $\Delta T = T_H - T_C$.

222 The dimensionless heat fluxes q^* are plotted in Fig. 1 (b). In the free-
 223 molecular flow regime, the dimensionless form q^* becomes independent of the
 224 rarefaction parameter δ_0 . Therefore, a flat region of the plots is an indication
 225 of the free-molecular flow regime. For all molar ratio conditions, the plots
 226 are almost flat and start to decrease at $\delta_0 > 1$, validating that the heat fluxes
 227 were measured from the free-molecular to near free-molecular flow regimes.
 228 The heat flux monotonically decreases with increase in the molar fraction
 229 of helium. In the numerical study using *ab initio* intermolecular potentials
 230 [5], the heat fluxes for pure components of helium and argon are reported
 231 as almost the same for whole range of the rarefaction parameter δ_0 ; while
 232 experimentally, they varies largely with the molar ratio. It is indicated that
 233 the variation with the molar ratio is due to a large variation in the thermal
 234 accommodation coefficient. Therefore, it is important to understand the
 235 gas-surface interaction to compare with the results in literatures. Relatively
 236 large scatterings of the plots in the small δ_0 region is due to the difficulty in
 237 measuring small heat transfer rates.

238 3.2. Thermal accommodation coefficient for pure gas species

239 Surface substrates of platinum were renewed from our previous measure-
 240 ment [14]. Thus, we measured the thermal accommodation coefficient for
 241 pure helium and argon to validate the values of this study by comparing
 242 them with our previous study and other studies. The thermal accommoda-
 243 tion coefficient was measured three times for each gas species to check the
 244 repeatability of the measurement.

Table 2: The dimensional heat fluxes q as a function of pressure p for He 0%:Ar 100%, He 50%:Ar 50% and He 100%:Ar 0% are tabulated.

He 0%:Ar 100%		He 50%:Ar 50%		He 100%:Ar 0%	
p (Pa)	q (W/m ²)	p (Pa)	q (W/m ²)	p (Pa)	q (W/m ²)
0.0125	0.315	0.00943	0.205	0.0135	0.359
0.0141	0.341	0.0126	0.239	0.0144	0.406
0.0148	0.362	0.0155	0.332	0.0153	0.414
0.0542	1.46	0.0537	1.21	0.0525	1.63
0.0556	1.39	0.0564	1.31	0.0530	1.59
0.0558	1.53	0.0593	1.31	0.0554	1.60
0.105	2.77	0.106	2.26	0.102	2.91
0.106	2.63	0.109	2.45	0.103	3.04
0.110	2.87	0.111	2.44	0.108	3.10
0.207	5.36	0.211	4.70	0.206	5.84
0.210	5.23	0.212	4.70	0.208	6.05
0.217	5.76	0.213	4.67	0.209	5.86
0.305	7.46	0.312	6.76	0.303	8.71
0.313	7.86	0.316	6.94	0.307	8.67
0.318	8.30	0.319	7.11	0.309	8.74
0.418	10.7	0.413	8.90	0.400	11.4
0.419	10.2	0.421	9.12	0.406	11.4
0.419	10.3	0.422	9.25	0.413	11.5
0.622	15.6	0.614	13.0	0.619	17.1
0.622	15.5	0.620	13.0	0.619	17.5
0.629	14.9	0.621	13.3	0.622	17.4
0.820	19.9	0.809	17.0	0.798	22.1
0.820	20.0	0.817	17.0	0.817	23.0
0.827	19.1	0.827	17.0	0.821	22.8
1.03	24.3	1.02	20.8	1.03	28.4
1.03	24.4	1.03	21.2	1.03	28.9
1.03	23.3	1.03	20.9	1.04	28.6
1.23	28.3	1.21	24.2	1.22	33.9
1.23	28.5	1.23	24.7	1.23	34.6
1.23	27.2	1.23	25.0	1.24	34.1

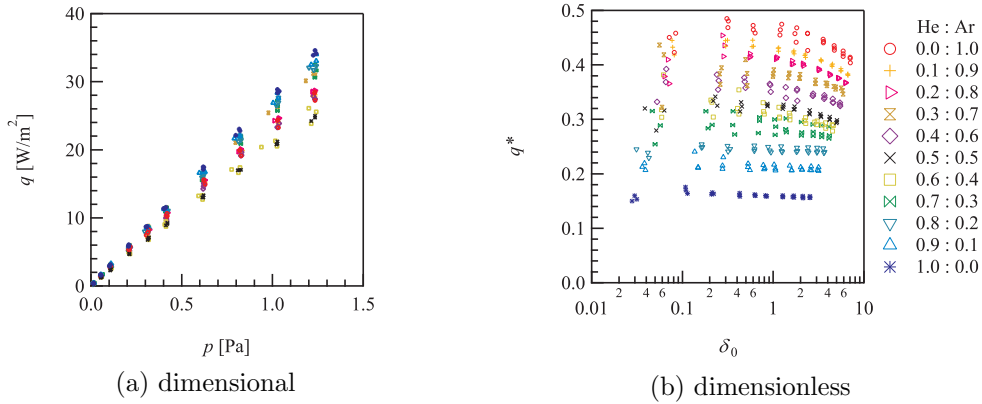


Figure 1: (Color online) All of the measured heat fluxes as a function of pressure for binary gas mixtures every 10% are plotted; (a) dimensional and (b) dimensionless forms.

245 The hot surface temperature T_H is approximately 340K and the cold sur-
 246 face temperature T_C is at a room temperature, and they are almost constant.
 247 The measured heat flux is fitted by Eq. (4) with Eqs. (3) and (7), whose
 248 expression in the continuum limit is corrected [21] from the previous measure-
 249 ment [14]. This correction considers the temperature distribution inside the
 250 chamber; while it was assumed to be uniform at T_C in the original analysis.
 251 In this study, by applying this correction, the obtained thermal accommoda-
 252 tion coefficient reduced from the original analysis by less than 0.1% for
 253 helium and 0.9% for argon. Therefore, this revision does not affect the result
 254 so much.

255 The results of the thermal accommodation coefficient on the platinum
 256 foil surface are plotted in Fig. 2. Three data points obtained in this study
 257 are plotted in the figure. It is difficult to distinguish each other especially in
 258 helium, Fig. 2 (a), because three measured values are aggregated in a single
 259 value, showing good repeatability of the measurement system. From the
 260 figure, measured thermal accommodation coefficients show good agreement
 261 with other data in literatures [18, 13, 30, 31]. In addition to that, measured
 262 values in this study give nearly the same with that in our previous study [21],
 263 showing good reproducibility. The reason for a little scattered data in argon
 264 could be cleanliness of the surface sample, where the time under vacuum
 265 before a measurement is not so long in this study. However, the discrepancy
 266 is negligibly small compared with scattering of data in literatures.

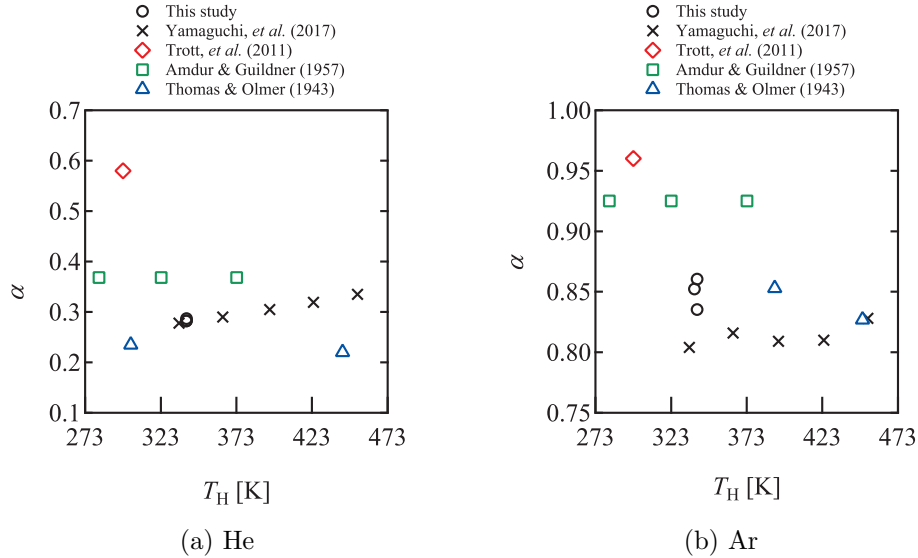


Figure 2: (Color online) The measured thermal accommodation coefficients on platinum for three measurements are plotted for pure gas species; (a) helium and (b) argon, and compared with published data in literatures [18, 30, 31].

267 3.3. Thermal accommodation coefficient for binary gas mixture

268 To obtain the thermal accommodation coefficient for a binary gas mixture,
 269 the measured heat flux is also fitted by Eq. (4). It is re-emphasized
 270 that the heat flux of a gas mixture in Eq. (3) is calculated from the mean
 271 molecular speed, $\bar{v}_{\text{mix}} = \sum \chi_i \bar{v}_i$. The heat flux was measured three times at
 272 each molar fraction, and the obtained thermal accommodation coefficients
 273 were averaged.

274 The results are shown in Fig. 3 and Table 3. Temperature conditions are
 275 also listed in the table.

276 To verify the use of Eq. (9) for the thermal conductivity of gas mixtures
 277 on calculating q_C in the continuum limit, the thermal accommodation coefficients
 278 are also obtained by using the thermal conductivity by *ab initio*
 279 simulations [25, 26, 27] and by Kestin, *et al.* (1984) [28] for the available
 280 molar ratio conditions. The results are plotted in Fig. 3 and Table 3. The
 281 discrepancies are very small, which is reasonable by considering that the
 282 heat flux was measured in the near free-molecular flow regime. Thus, it is
 283 reasonable to use Eq. (9) for the the thermal conductivity.

284 To compare the results with the theoretical value in the free-molecular

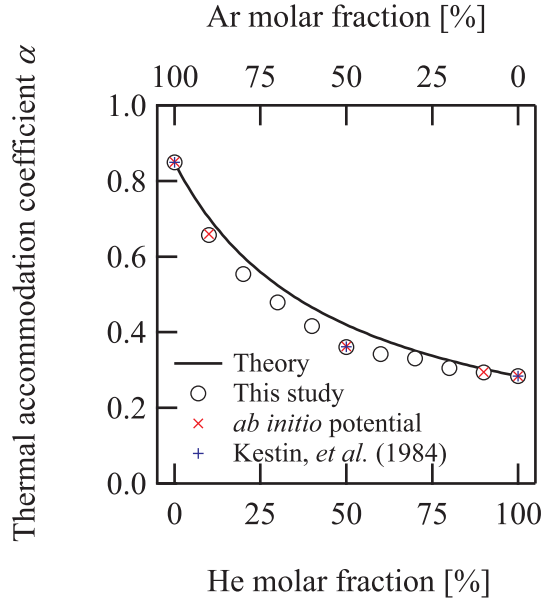


Figure 3: The measured thermal accommodation coefficients as a function of a molar fraction are plotted by markers. The thermal conductivity is changed for the calculation of q_C to the value obtained from *ab initio* simulations [25, 26, 27] and by Kestin, *et al.* (1984) [28]. They are compared with the theoretical curve calculated by Eq. (2) using thermal accommodation coefficients of each component obtained in this study.

285 flow regime, Eq. (2), a curve is calculated by using the thermal accommo-
 286 dation coefficients for pure gas components, helium and argon, measured in
 287 this study, which is also plotted in the figure. Since the theoretical curve
 288 for the binary gas mixture is calculated from the measured values for helium
 289 and argon, data points of helium 0% (argon 100%) and helium 100% (ar-
 290 gon 0%) are exactly on the curve. The data points are almost explained by
 291 the theoretical concave curve. This slightly curved tendency connecting two
 292 thermal accommodation coefficients of pure gas components is also obtained
 293 by a molecular dynamics study using the pair of argon - xenon (Fig. 7 in
 294 [15]). Thus, it is reasonable to use the theoretical relation to evaluate the
 295 thermal accommodation coefficient for a binary gas mixture.

296 The dimensionless heat flux q^* monotonically decrease with increasing in
 297 the fraction of helium in a gas mixture in Fig. 1 (b), which is considered to
 298 be the effect of the gas-surface interaction. By using the obtained thermal

Table 3: The measured thermal accommodation coefficients with temperature conditions. The thermal conductivity is changed for the calculation of q_C to the value obtained from *ab initio* simulations [25, 26, 27] and by Kestin, *et al.* (1984) [28].

He	Ar	T_H [K]	T_C [K]	α	<i>ab initio</i>	Kestin, <i>et al.</i>
0%	100%	339.6	293.1	0.8492	0.8500	0.8492
10%	90%	340.6	293.0	0.6571	0.6594	
20%	80%	339.9	291.8	0.5535		
30%	70%	344.7	293.1	0.4788		
40%	60%	338.6	292.7	0.4165		
50%	50%	334.2	292.1	0.3610	0.3620	0.3619
60%	40%	333.8	292.7	0.3419		
70%	30%	340.5	292.9	0.3304		
80%	20%	339.9	291.9	0.3047		
90%	10%	339.6	292.3	0.2940	0.2942	
100%	0%	340.0	293.1	0.2838	0.2837	0.2837

299 accommodation coefficients, the measured heat fluxes are compared with the
300 numerical results in literatures [5, 21]. In the free-molecular flow regime, the
301 heat flux is proportional to the thermal accommodation coefficient as in Eq.
302 (3). Thus, the dimensionless heat flux q^* is divided by the thermal accom-
303 modation coefficient α to exclude the effect from the gas-surface interaction
304 for direct comparison. q^*/α are plotted in Fig. 4 (a). It is worth noting that
305 the geometrical configuration is the parallel plates in Ref. [5], whereas the
306 concentric spheres are employed in Ref. [21]. The characteristic length R_0
307 is the distance between the surfaces. From the figure, the results of helium
308 and argon give almost the same values including the experimental results.
309 Therefore, it is verified that the variation of the dimensionless heat flux in
310 Fig. 1 (b) results from the gas-surface interaction. In the free-molecular flow
311 regime with small δ_0 , all of the results show quite good agreement. However,
312 the numerical results of the parallel plates start to decrease much smaller δ_0
313 condition. This could be because of the difference in the geometrical config-
314 uration. Therefore, the characteristic length is changed to $R_0 = R_H$ in Fig.
315 4 (b). The curves show good agreements; while, there is a small discrepancy
316 in large δ_0 region, indicating the difference in the geometrical configuration
317 still remains. The heat flux is also compared with the result of a gas mixture
318 (He 50%:Ar 50%) in Ref. [5]. From the figure, the reasonable agreements are
319 observed even for the results of the gas mixture, and it is validated that the

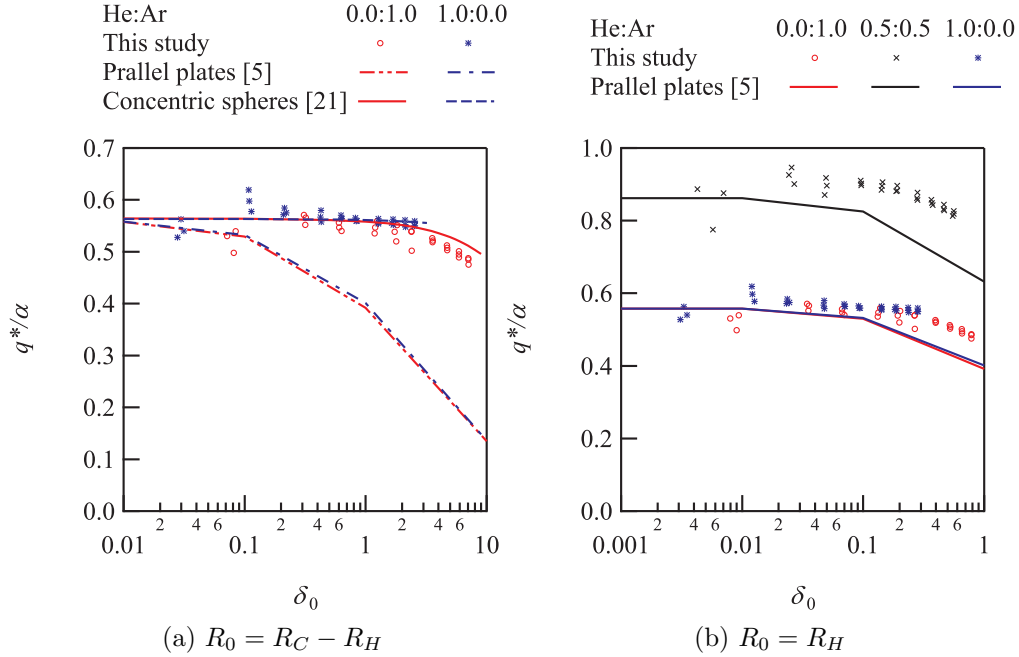


Figure 4: (Color online) The dimensionless heat flux divided by the obtained thermal accommodation coefficient is plotted for pure gas species and a gas mixture (He 50%:Ar 50%), and compared with published data in literatures for the parallel plates geometry [5] and for the concentric spheres geometry [21]; (a) the characteristic length $R_0 = R_C - R_H$ and (b) $R_0 = R_H$.

320 dimensionless heat flux for the mixture is larger than that for a single gas.
 321 However, if we investigate Fig. 3 carefully, there is a small discrepancy
 322 between the plots and curve. It is interesting to point out that all the plots
 323 lie below the curve, even though there is a scattering of data due to a mea-
 324 surement error. The plots of pure gas components, which are at both ends
 325 of the curve, also suffer from the measurement error as well as the plots for
 326 gas mixtures between them; nevertheless, all the plots deviate in the same
 327 side, smaller than the curve. Therefore, it is natural to consider that the
 328 smaller thermal accommodation coefficients are owing to the mixing of two
 329 gas components, the nature of gas mixture. In addition to that, the thermal
 330 accommodation coefficient showed about 5% reduction in maximum by the
 331 revision of the empirical function, Eq. (4), in our recent study [21]. There-
 332 fore, even by the revision of the empirical function to derive the thermal

333 accommodation coefficient, the thermal accommodation coefficient will de-
 334 crease, resulting in enlarging the discrepancy from the curve. In our previous
 335 study [16], the viscous slip coefficient was measured for a binary gas mixture
 336 as a function of a molar fraction. The viscous slip coefficient was a convex
 337 function of a molar fraction, whose shape was similar to that obtained by
 338 numerical simulations qualitatively; however, there were discrepancies quan-
 339 titatively. It was considered that the difference in the tangential momentum
 340 accommodation coefficients for pure gas components between our measure-
 341 ment and the numerical studies resulted in the discrepancies. However, it was
 342 difficult to explain our measured results from an extrapolation of the numer-
 343 ical results only by the difference in values for pure gas components, and it
 344 was necessary to assume the accommodation coefficient as a function of a
 345 molar fraction for fitting our results by the numerical results. The tangential
 346 momentum accommodation coefficient was estimated as a concave function of
 347 a molar fraction. From this result, the accommodation coefficient in the slip
 348 flow regime was implied to vary with a molar fraction. Since the thermal ac-
 349 commodation coefficient measured in this study was analyzed from the data
 350 not only in the free-molecular but also in the early transitional flow regime,
 351 it is not strange to assume that our measured thermal accommodation co-
 352 efficients suffer from this characteristics. Thus, the ratio of the measured
 353 thermal accommodation coefficient to the theoretical one is calculated and
 354 plotted in Fig. 5. The error bar is calculated from the standard error for
 355 three measurements. The detailed assessment of uncertainties could be re-
 356 alized by the uncertainty quantification technique [32]. From the figure, a
 357 concave tendency of the thermal accommodation coefficient can be observed.
 358 Just for an indication of tendency, a quadratic function, $y = 1 + Ax(1 - x)$,
 359 which gives unity at both ends of 0% and 100% in the molar fraction, is also
 360 plotted by a dashed line, where A is a constant and x is the molar fraction.
 361 It is important to note that the ratio is almost unity and the smallest value
 362 is even larger than 0.85. From the accuracy of our simplified measurement
 363 system, the only conclusion is that the measured thermal accommodation
 364 coefficient was almost the same with the theoretical one; however, it is sug-
 365 gested that the thermal accommodation coefficient decreases by mixing two
 366 gas components in the slip flow regime. There is, of course, a possibility of
 367 the effect of the empirical relation used in this study, since it did not fit well
 368 to the numerical results by the S-model kinetic equation [21]. Therefore, the
 369 measurements of the temperature jump coefficient for a binary gas mixture
 370 is needed [33], which will reveal this characteristics directly.

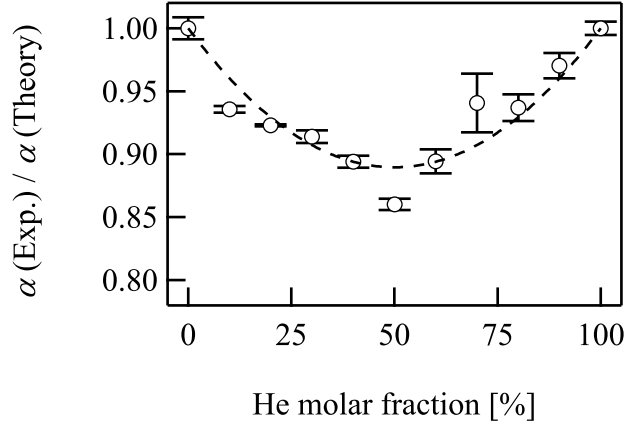


Figure 5: The ratio of the measured thermal accommodation coefficient to the theoretical value obtained by Eq. (2) are plotted by markers. Dashed line is a quadratic function $y = 1 + Ax(1 - x)$ just for an indication of the shape. Error bars indicate the standard errors.

371 4. Conclusions

372 The conductive heat flux through a rarefied binary gas mixture of he-
 373 lium and argon is measured as a function of molar fraction every 10%. The
 374 dimensionless heat flux monotonically decreases with increasing the molar
 375 fraction of helium. This is coming from the large difference in the thermal
 376 accommodation coefficient between helium and argon.

377 Then, the thermal accommodation coefficients were obtained for binary
 378 gas mixtures. The measured thermal accommodation coefficients for pure gas
 379 components showed good agreements with our previously reported values
 380 and data in literatures, showing good repeatability and reproducibility of
 381 our measurement system. For a binary gas mixture, the measured thermal
 382 accommodation coefficient was well explained by the theoretical curve derived
 383 by considering a superposition of an independent heat transfer by each gas
 384 component in the free-molecular flow regime.

385 If we investigate the results carefully, the thermal accommodation coef-
 386 ficient was below the theoretical curve, inferring a concave variation of the
 387 coefficient as a function of a molar fraction. This characteristics would be
 388 similar to those observed in the viscous slip coefficient for a binary gas mix-
 389 ture.

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